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6.2 Some basic concepts 6.2.1 Thermodynamic system

A thermodynamic system is a certain portion of the universe selected for the purpose of investigation and is thus distinct, being set apart from everything else, and is essentially macroscopic. The system may be a gas such as air, a vapour such as steam, a vapour in contact with its liquid such as liquid ammonia and amonia vapour, a mixture such as air and gasoline vapour etc. In addition to these systems relevant to engineering in particular, there may be such thermodynamic systems as a stretched wire, electric capacitors, thermocouples, magnetic materials, surface films and electric cells which are of more relevance in physics. Plainly, a thermodynamic system is perceptible by our senses.

A system may be simple or complex. It may also be homogeneous or heterogeneous where each component can exist in different phases. A gas enclosed in a cylinder fitted with a frictionless gas-tight piston is a simple homogeneous system, but a phenol-water mixture is an example of a complex heterogeneous system. Whatever be the system, it is always finite.

6.2.2 Surroundings and boundary

By surroundings of a system is meant everything outside it that can influence its behaviour. The boundary of a system is the envelope that encloses the system and thereby separates it from

its surroundings. This is illustrated in Fig.6.1. The importance of the boundary lies in visualising the system distinctly and it may be real or imaginary.

The boundary of a system may or may not allow it to interact with its surroundings. The boundary that does not allow any exchange of matter and energy between the system and its surtoundings is called an isolating boundary. A system bounded by an isolating boundary is termed an isolated system and is not of much importance thermodynamically. A system with a boundary that permits exchange of matter between the system and the surroundings is called an open system. The system, however, is said to be a closed system if its boundary allows exchange of energy, but prevents exchange of matter between the system and the surroundings. A closed system, however, is not an isolated system. The exchange of energy between the system and the surroundings may take place

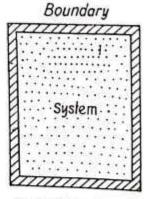


Fig.6.1 Thermodynamic system and boundary

either thermally or by doing work on the system. The surface or boundary that prevents thermal interaction with surroundings is called adiabatic, and the system is thermally isolated. If, however, heat exchange can occur through the boundary, it is said to be diathermic. A system with a diathermic boundary will necessarily be in thermal contact with the surroundings.

6.2.3 State of a system and and thermodynamic variables

In mechanics, the motion of a system is described by solving its equation of motion the solution giving information relating to time-variation of the position vector associated with the system. In a like manner, in thermodynamics, the state of a system at any instant is represented by its condition at that instant, the condition being completely specified by a set of experimentally measurable quantities, called variables of state or thermodynamic variables. For example, to a stretched wire correspond the thermodynamic variables tension and length; for a surface film the variables are the surface tension and the area; for a capacitor they are e.m.f and the charge; for a hydrostatic system the variables are pressure and volume; and so on. The thermodynamic variables are also termed thermodynamic coordinates. A particularly simple condition of state of a system is the equilibrium state in which the variables specifying the state are time-independent, that is, do not change with time and are reproducible (Read Art. 6.2.4).

The thermodynamic variables may be of two different kinds: intensive and extensive, Intensive variables of a system in a given state are those which are independent of its mass or the number of particles. Extensive variables, on the other hand, are proportional to the mass or, to the number of particles in the system. Tension, surface tension, e.m.f., in the above examples are intensive variables; length, area, charge are extensive variables. Pressure and temperature are intensive, while volume is an extensive variable.

We give below the thermodynamic variables of some important thermodynamic systems.

	System	Intensive variable	Extensive variable
1.	Hydrostatic system	pressure, p	volume, V
2.	Surface film	surface tension, S	area, A
3.	Stretched wire	tension, F	length, L
4.	Electric cell	potential difference, E	charge, Z
5.	Paramagnetic substance	magnetic intensity, H	dipole moment, M

If u be a macroscopic parameter characterizing the state of a homogeneous system and if the system be divided, by a partition, into two parts or sub-systems having y-values y1 and y2 respectively, then

- (i) y is extensive, if $y = y_1 + y_2$
- (ii) y is intensive, if $y = y_1 = y_2$

If x, y be two arbitrary intensive variables, then $xy, y/x, \partial y/\partial x, x + y$ are also intensive. If X. Y are two arbitrary extensive variables, then X + Y is also extensive, but $Y/X, \partial Y/\partial X$ are intensive. Similarly, if x is intensive and X extensive, then $xX, X/x, \partial X/\partial x$ will be extensive. Extensive variables when referred to their specific values, that is, the value per unit mass of the system are known as specific variables. Interestingly, a specific variable corresponding to any extensive variable becomes, by definition, mass-independent and is thus an intensive variable.

It is customary to designate the extensive variables by capital letters and their specific values by the corresponding small letters. For example, if V be the total volume of the system, will represent the specific volume.

6.2.5 Thermodynamic processes

A system is said to undergo a thermodynamic process, when the values of its thermodynamic variables or coordinates change from one equilibrium state to another.

To analyse a thermodynamic process, the variation of one thermodynamic variable is plotted with respect to another and the plot is known as indicator diagram (Fig. 6.2). On such a diagram the state of a system is determined uniquely by a point. And a process means a line connecting a series of such points. In the adjacent diagram, the initial state of the system is represented by the point (p_1, V_1) , the system undergoes an expansion, and the final state of the system is defined by the point (p_2, V_2) . The process is represented by the curve shown in Fig. 6.2. Indicator diagram here is p-V diagram.

It is worth noting that joining the initial and final points by a line has an important implication in that the (p₁,v₁)

Fig.6.2 An indicator diagram

points by a line has an important implication in that the intermediate states are also equilibrium states. And this at once imposes some restrictions as regards the evolution of the thermodynamic process. Since the intermediate states are uniquely defined, they being all equilibrium states, a thermodynamic process can be made to retrace its original path to reach back the initial state. How to execute the retracting (that is, what conditions need be satisfied) however is a separate story. If retracing is possible, the process is said to be reversible, if not it is said to be irreversible. All natural processes are irreversible. We shall discuss reversibility later in greater detail.

The concept of reversibility facilitates the mathematical formulation of thermodynamics. But how to analyse a natural process within the framework of such a formulation? The problem has been overcome by introducing what may termed a quasi-static process, discussed in a separate sub-section (6.2.9) that follows. Essentially, when a process is carried out extremely slowly such that every state through which the system passes departs only infinitesimally from equilibrium, the process is said to be quasi-static. A quasistatic process is thus a succession of thermodynamic equilibrium states. For finite departures from equilibrium, the process becomes non-quasistatic where system variables do not define the states through which the system passes. They cannot also describe the processes the system undergoes. Consequently, a non-quasistatic process cannot be represented by a line on the indicator diagram.

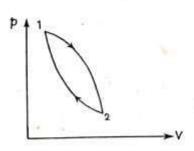


Fig.6.3 A cyclic process represented on p-V diagram

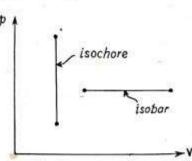


Fig. 6.4 An isochoric and an isobaric process on p-V diagram

If the series of processes undergone by a system leads to its reversal back to the initial state, the series constitutes a cyclic process and is represented by a closed path on the indicator diagram (Fig.6.3). Some processes, again, are so characterised that a thermodynamic coordinate

of the system remains constant throughout. A process in which volume is so kept constant is called *isochoric*, represented on the indicator diagram by a straight line parallel to the p-axis (Fig.6.4). A process where pressure remains constant is called *isobaric* represented by a line parallel to V-axis (Fig.6.4) on the indicator diagram. Similarly, a process in which temperature remains constant is called *isothermal* and one in which there is no thermal interaction (exchange) taking place between the system and its surroundings is called *adiabatic*.

6.2.6 Change in state in an infinitesimal process

If a system undergoes a small change of state in making transition from an initial equihibrium state to another state very near to the former, all the coordinates in general, undergo small changes. If the change in volume is small compared to V, but quite large compared to the volume occupied by few molecules, we may denote it by differential dV. Since V is a macroscopic coordinate, 'dV' will be meaningful when it is large enough to accommodate quite a large number of molecules. Similarly, change of pressure is small compared to p, but large in the sense of moelcular fluctuation, and is denoted by dp.

So the infinitesimals in thermodynamics are characterised as small compared to the quantity itself but large with reference to the behaviour of few molecules. As already stated in kinetic theory, it makes no sense in stating the temperature, volume and pressure of a few molecules.

It is obvious from the foregoing that the equation of state can be used to calculate the physical parameters of the system like volume expansivity, bulk modulus etc. as has been shown in Art. 6.2.7.

6.3 First law of thermodynamics : Background and formulation

The first law of thermodynamics is a mathematical formulation of the quantitative aspect of the law of conservation and transformation of energy.

The classical experiments of Davy, Rumford, Mayer and Joule led to the discovery of the equivalence of heat and work. To arrive at the formulation of the first law we consider a system completely surrounded by adiabatic envelope but coupled to the surroundings so that work may be done. It is possible to visualise many adiabatic paths connecting the initial and the final states of the system. They are accompanied by non-static dissipative processes. The amount of work done along different adiabatic paths indicated by indirect experiments is the same along all the paths.

The generalised statement of this result is known as the first law of thermodynamics.

If a thermally insulated system is caused to change from an initial state to a final state, the amount of work done is the same for all adiabatic paths connecting the state.

The dependence of the adiabatic work on only the initial and final states of the system and its invariance with respect to the path of transformation enables us to invoke a single-valued function of state variables of the thermodynamic system, such that the value of the function in the final state minus that at the initial state is equal to the adiabatic work. This function is the internal energy function U of the system and is therefore.

$$-W_{i\rightarrow f}|_{ad} = U_f - U_i$$

the negative sign is in conformity with the signature of work.

For the work done by the system, $W_{i \to f}$ is positive so that $U_f < U_i$, that is, the internal energy of the system decreases, consistent with the principle of conservation of energy. Similarly, when work is done upon the system, $W_{i \to f}$ is negative, $U_f > U_i$, the internal energy of the system increases as expected from the principle of conservation of energy. So in order to measure the change in internal energy of a system, experiments involving adiabatic work must be performed. In the laboratory, such experiments are difficult to carry out.

6.4 Mathematical formulation of the first law

The equation (i) of the previous Art. 6.3 may be written for an infinitesimal process as

$$d\bar{Q} = dU + dW \tag{i}$$

For a chamical system undergoing quasi-static process

$$\bar{d}Q = dU + pdV \tag{ii}$$

For a system of streched wire (elastic system)

$$\bar{d}Q = dU - FdL \tag{iii}$$

F and L being the tension and length respectively.

For a magnetic system

$$\bar{d}Q = dU - HdM \tag{iv}$$

where H is the magnetic intensity and M, the magnetic moment.

From the foregoing analysis, we may conclude that the first law of thermodynamics is more than a statement of the principle of conservation of energy. The formulation leads to the existence of 'internal energy' function U as a single -valued function of state variables specifying the configuration of the thermodynamic system and defines 'heat' as energy during transit by virtue of temperature difference.

6.5 Applications of the fir I law

The first law of thermodynamics has been successfully applied to a number of physical and chemical processes, both in the laboratory and outside in nature. We shall discuss in this section some of those applications.

Heat capacities: For an infinitesimal quasistatic process performed by one gm. mol. of a chemical system

$$\bar{d}Q = dU + pdV \tag{i}$$

IQ being not an exact differential, but dU and dV are; U and V being functions of coordinates lefining the equilibrium state of the system.

Let us assume that U is a function of two independent variables T and V of the system.

$$U = U(T, V)$$

$$dU = \left(\frac{\partial U}{\partial T}\right)_V dT + \left(\frac{\partial U}{\partial V}\right)_T dV$$
(ii)

Substituting (ii) in (i), we obtain

$$\bar{d}Q = \left(\frac{\partial U}{\partial T}\right)_V dT + \left\{p + \left(\frac{\partial U}{\partial V}\right)_T\right\} dV$$
 (iii)

For a process where V = const., dV = 0 (isochoric process). So, from (iii),

$$C_V = \left(\frac{\bar{d}Q}{dT}\right)_V = \left(\frac{\partial U}{\partial T}\right)_V$$
 (iv)

Cv being the heat capacity at constant volume. Since we have taken one gm. mol., it is also the molar specific heat at constant volume.

Similarly, expressing both U and V as functions of temperature T and pressure p, for an infinitesimal quasistatic process

$$\bar{d}Q = \left(\frac{\partial U}{\partial T}\right)_{p} dT + \left(\frac{\partial U}{\partial p}\right)_{T} dp + p \left\{ \left(\frac{\partial V}{\partial T}\right)_{p} dT + \left(\frac{\partial V}{\partial p}\right)_{T} dp \right\} \tag{v}$$

For a process where p = const., dp = 0 (isobaric process). So from (v), we get

$${}'C_p = \left(\frac{\bar{d}Q}{dT}\right)_p = \left(\frac{\partial U}{\partial T}\right)_p + p\left(\frac{\partial V}{\partial T}\right)_p$$
 (vi)

where Cp is the heat capacity at constant pressure. Here also it is the molar specific heat at constant pressure.

For an isothermal quasistatic process, dT = 0. We obtain from (iii) and (v)

$$\left(\frac{\bar{d}Q}{dV}\right)_T = p + \left(\frac{\partial U}{\partial V}\right)_T = L_V, \text{say}.$$
 (vii)

and
$$\left(\frac{\bar{d}Q}{dp}\right)_T = \left(\frac{\partial U}{\partial p}\right)_T + p\left(\frac{\partial V}{\partial p}\right)_T = L_p$$
, say. (viii)

We define Lv, the latent heat of expansion, as the amount of heat required to increase the volume of unit mass (or 1 gm. mol.) of the substance quasi-statically, temperature remaining constant. Similarly, L_F is the heat required to increase the pressure of unit mass of the system

at constant temperature. From (vii) and (viii), using (i), we respectively have

$$\bar{d}Q = C_V dT + L_V dV \tag{ix}$$

$$dQ = C_p dT + L_p dp$$
and
$$dQ = C_p dT + L_p dp$$
(x)

Difference between specific heats : $C_p - C_V$: From eq. (iii) we obtain

reen specific heats:
$$C_p = \left(\frac{dQ}{dT}\right)_p = \left(\frac{\partial U}{\partial T}\right)_V + \left\{p + \left(\frac{\partial U}{\partial V}\right)_T\right\} \left(\frac{\partial V}{\partial T}\right)_p$$
 (xi)

$$= C_V + \left\{ p + \left(\frac{\partial U}{\partial V} \right)_T \right\} \left(\frac{\partial V}{\partial T} \right)_p$$

$$C_p - C_V = \left\{ p + \left(\frac{\partial U}{\partial V} \right)_T \right\} \left(\frac{\partial V}{\partial T} \right)_p$$
(xii)

The first term in the bracket on the right gives the amount of work done in pushing back the surroundings at constant pressure, while the second term tells how the internal energy changes with volume, being associated with the work done against intermolecular forces.

Introducing the volume expansivity, $\beta=\frac{1}{V}\left(\frac{\partial V}{\partial T}\right)_p$, eq. (xii) may be written as

$$C_p - C_V = \left\{ p + \left(\frac{\partial U}{\partial V} \right)_T \right\} \beta V = L_V \beta V$$
 (xiii)

where L_V is the latent heat of expansion, already defined.

Special cases - For a perfect gas, pV = RT so that $p(\partial V/\partial T)_p = R$. Also, since there is no intermolecular attraction, its internal energy is wholly kinetic and remains unchanged in free expansion (see later). Thus, $(\partial U/\partial V)_T = 0$. (xiv)

expansion (see later). Thus,
$$(C_p - C_V = R)$$
 (xiv)

For n gm. mol, it takes the form $(C_p - C_V = R)$ (xiva)

For n gm. mol, it takes the form

$$C_p - C_V = nR$$
 (xiva)

This is known as Mayer's relation. For a real gas, however, the internal energy changes with volume, for work has to be done against the intermolecular forces. Taking the van der Waals' model of real gases,

$$(p+a/V^2)(V-b) = RT$$

We shall see later that $(\partial U/\partial V)_T = a/V^2$, so that from (xii)

$$C_p - C_V = \frac{RT}{V - b} \left(\frac{\partial V}{\partial T} \right)_p$$
 (xv)

Differentiating van der Waals' equation

$$\left\{p+\frac{a}{V^2}-(V-b)\frac{2a}{V^3}\right\}\left(\frac{\partial V}{\partial T}\right)_p=R$$

$$\left(\frac{\partial V}{\partial T}\right)_{p} = \frac{R}{p + \frac{a}{V^{2}} - \frac{2a}{V^{3}}(V - b)} = \frac{R(V - b)}{RT - \frac{2a}{V^{3}}(V - b)^{2}}$$

$$\therefore \frac{1}{V-b} \left(\frac{\partial V}{\partial T} \right)_p = \frac{1}{T \left\{ 1 - \frac{2a}{RTV^3} (V-b)^2 \right\}} = \frac{1}{T} \left\{ 1 + \frac{2a}{RTV^3} (V-b)^2 \right\}$$

expanding binomially and retaining terms up to first order in a which is very small.

... From (xv), we thus obtain

$$C_p - C_v = R \left\{ 1 + \frac{2a}{RTV^3} (V - b)^2 \right\}$$
 (xvi)

Comparing (xvi) with (xiv), it is observed that the difference is more for a real gas. For large molecular separations, the second term in the bracket of (xvi) is ignorable and the result is one with the perfect gas. Fig.6.7 shows that

the plot of $(C_p - C_v)/R$ vs. V for a van der Waals' gas. It is observed that the ideal gas Cp-Cv value is attained only at a large value of V.

Free expansion: In their experiments on free expansion of a gas, Joule and Mayer observed that the temperature of the gas remains unaltered.

In the free expansion, a gas is kept in one part of the vessel by a partition, the other part evacuated. On removing the partition. the gas is allowed to expand against vacuum and fills the whole vessel (Fig.6.8). Free expansion is a non-static process and takes place under thermal isolation. Since the gas expands against vacuum, no external work is done by the gas.

Fig. 6.7 Plot of $(C_p-C_v)/R$ vs. V of a van der Waals' gas

$$\vec{d}Q = 0$$
, $\vec{d}W = 0$

Hence, from the first law, dU = 0. If U is assumed to depend on T and V,

$$dU = \left(\frac{\partial U}{\partial T}\right)_{v} dT + \left(\frac{\partial U}{\partial V}\right)_{T} dV$$

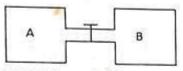


Fig.6.8 Joule expansion (Free expansion)

Since dT = 0 (as was observed by Joule and Mayer), and also dU = 0, we obtain

$$\left(\frac{\partial U}{\partial V}\right)_T = 0$$
, since $dV \neq 0$.

Similarly, taking U as a function of T and p, we would get

$$\left(\frac{\partial U}{\partial p}\right)_T = 0$$

The internal energy of such a system depends on temperature T only and is independent of p and V. The results on change of temperature are valid only for an ideal gas, although Joule and Mayer performed experiments on real gases. The thermometers used by them for measurement of temperature after free expansion were of low precision and they considered the gases used to obey the Boyle's and Charles' laws (ideal gases).

6.9 Work in quasi-static processes

1. (a) Isothermal expansion or compression of an ideal gas, the initial volume V changing finally to V_{ℓ} . The work

$$\begin{split} W_{iso} &= \int_{V_i}^{V_f} p dV = \int_{V_i}^{V_f} \frac{nRT}{V} dV \quad (\because pV = nRT) \\ &= nRT \int_{V_i}^{V_f} \frac{dV}{V} \\ &= nRT \ln \frac{V_f}{V_i} \end{split}$$

2. Adiabatic expansion or compression of an ideal gas, the initial volume Vi changing finally to V_f . The work W_{ad} is given by

$$W_{\text{ad}} = \int_{V_{i}}^{V_{f}} p dV = k \int_{V_{i}}^{V_{f}} V^{-\gamma} dV \qquad (\because pV^{\gamma} = k \text{ (const.)})$$

$$= \frac{k}{\gamma - 1} \left[V^{1-\gamma} \right]_{V_{f}}^{V_{i}} = \frac{k}{\gamma - 1} \left(V_{i}^{1-\gamma} - V_{f}^{1-\gamma} \right)$$

$$= \frac{p_{i}V_{i} - p_{f}V_{f}}{\gamma - 1} \qquad (\because p_{i}V_{i}^{\gamma} = k = p_{f}V_{f}^{\gamma})$$

$$= \frac{R}{\gamma - 1} \left(T_{i} - T_{f} \right) \qquad (\because p_{i}V_{i} = RT_{i}; p_{f}V_{f} = RT_{f})$$
(iii)

(111)

The relations (ii), (iii) and (iv) constitute the different expression for the adiabatic work done by an ideal gas. The corresponding expressions for a van der Waals' gas are given elsewhere.

6.2.7 Some deductions from equation of state

The equation of state in the parametric form may be used to study the behaviour of a system under different conditions.

Consider a hydrostatic system with the equation of state

$$f(p, V, T) = 0$$
or, $p = p(V, T)$

$$\therefore dp = \left(\frac{\partial p}{\partial V}\right)_T dV + \left(\frac{\partial p}{\partial T}\right)_V dT \tag{i}$$

where dp is an infinitesimal change in p. The $(\partial p/\partial V)_T$ expresses the rate of change of pressure with volume when the temperature is constant, that is, in an isothermal process; $(\partial p/\partial T)_v$ is the rate of change of pressure with temperature when the volume is kept constant, that is, for an isochoric process.

For an isobaric process, dp = 0 and eq. (i) becomes

$$\left(\frac{\partial p}{\partial T}\right)_{V} = -\left(\frac{\partial p}{\partial V}\right)_{T} \left(\frac{\partial V}{\partial T}\right)_{p} \tag{ii)}$$

To attribute physical meaning to (i), we note that volume expansivity β of a material, in an isobaric process, is given by

 $\beta = \frac{1}{V} \left(\frac{\partial V}{\partial T} \right)_{p} \tag{iii}$

and isothermal elasticity, E_T , reciprocal of isothermal compressibility K_T , of a material is defined as

$$E_T = \frac{1}{K_T} = -V \left(\frac{\partial p}{\partial V}\right)_T$$
 (iv)

Combining (ii), (iii) and (iv), we obtain

$$\left(\frac{\partial p}{\partial T}\right)_{V} = \frac{\beta V E_{T}}{V} = \beta E_{T} \tag{9}$$

Thus, for a given change in temperature, the increase in pressure will be more when expansivity and also elasticity are large (compressibility small) and conversely.

Using (v), we get from (i)

$$dp = -\frac{E_T}{V}dV + \beta E_T dT \tag{vi}$$

For an isochoric process, (vi) reduces to

$$dp = \beta E_T dT \tag{vii}$$

For a finite change in presure, the change in temperature is

$$\int_{p_1}^{p_2} dp = \int_{T_1}^{T_2} \beta E_T dT$$
 or, $p_2 - p_1 = \beta E_T (T_2 - T_1)$ (viii)

assuming β and E_T constant in the temperature range considered.

Thus, if β and E_T are experimentally determined, eq. (viii) provides the final pressure, in an isochoric process, for a given rise in temperature.